

(Following Paper ID and Roll No. to be filled in your
Answer Books)

Paper ID : 140662

Roll No. 1303240829

B.TECH.

Theory Examination (Semester-VI) 2015-16

FINITE ELEMENT METHODS

Time : 3 Hours

Max. Marks : 100

Note : Assume any missing data suitably.

Section-A

1. Attempt all parts. All parts carry equal marks.

(2×10=20)

- (a) What are the limitations of Galerkin formulation?
- (b) Write down the stiffness matrix for 2D beam element.
- (c) What do you mean by convergence in finite element analysis?
- (d) Why polynomial shape functions are preferred?
- (e) Specify stress and strain tensors for plane stress case.

(1)

P.T.O.

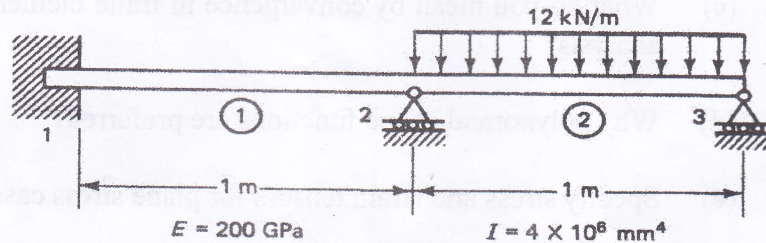
- (f) What are the advantages of expressing displacement field in Natural co-ordinates than generalized co-ordinates?
- (g) Write down the shape functions for four noded rectangular elements.
- (h) Write the shape function for constant strain triangle by using polynomial function.
- (i) What are the conditions for a problem to be axisymmetric?
- (j) What are the steps involved in finite element modeling?

Section-B

Note : Attempt any five questions from this section.

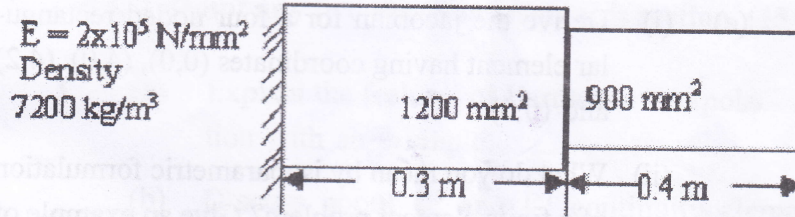
(10×5=50)

- (a) Determine the vertical deflection at the midpoint of the distributed load for the beam shown in Fig.

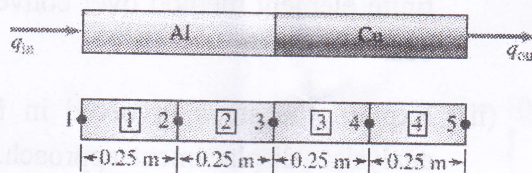


(2)

- (b) Consider the axial vibrations of a steel bar shown in the Fig.:
- Develop global stiffness and mass matrices.
 - Determine the natural frequencies.



- (c) The circular rod depicted in figure has an outside diameter of 60 mm, length of 1 m, and is perfectly insulated on its circumference. The left half of the cylinder is aluminum, for which $k_x = 200 \text{ W/m}^\circ\text{C}$ and the right half is copper having $k_x = 389 \text{ W/m}^\circ\text{C}$. The extreme right end of the cylinder is maintained at a temperature of 80°C , while the left end is subjected to a heat input rate 4000 W/m^2 . Using four equal-length elements, determine the steady-state temperature distribution in the cylinder.



- (d) For a square, isotropic elastic body of thickness 'h', the displacement are approximated by:

(3)

P.T.O.

$$u(x,y) = y(1-x)u_1 + x(1-y)u_2$$

$$v(x,y) = 0$$

Assuming plane stress condition, derive the stiffness matrix for the unit dimensioned square.

- (e) (i) Derive the jacobian for a four noded rectangular element having coordinates (0,0), (4,0), (4,2) and (0,2).
- (ii) What do you mean by isoparametric formulation of a finite element problem? Give an example of real field problem where super- parameteric elements can be used and why ?
- (f) Evaluate the integral

$$I = \int_{-1}^1 \frac{r^2 - 1}{(r + 3)^2} dr$$

using Gaussian integration with one, two, and three integration points.

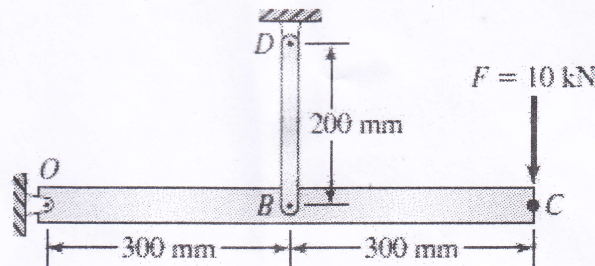
- (g) (i) Discuss the advantages and disadvantages of finite element method over conventional methods.
- (ii) Explain the steps involved in finite element analysis - displacement approach.
- (h) (i) What are the convergence and compatibility requirements? Discuss in detail.

- (ii) Differentiate conforming and non-conforming elements.

Section-C

Note: Attempt any two questions from this section. (15×2=30)

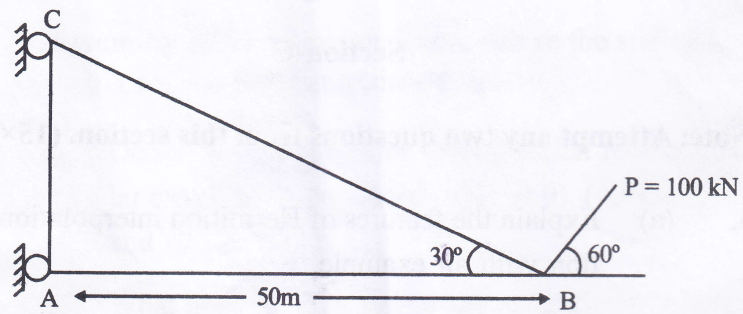
3. (a) Explain the features of Hermitian interpolation function with an example.
- (b) Discuss about C^0 and C^1 continuity elements in detail.
4. In Figure, beam OC is supported by a smooth pin connection at O and supported at B by an elastic rod BD , also through pin connections. A concentrated load $F = 10$ kN is applied at C . Determine the deflection of point C and the axial stress in member BD . The modulus of elasticity of the beam is 207 GPa (steel) and the dimensions of the cross section are 40 mm × 40 mm. For elastic rod BD , the modulus of elasticity is 69 GPa (aluminum) and the cross-sectional area is 78.54 mm².



(5)

P.T.O.

5. Develop the stiffness matrix & determine nodal displacement for given truss. Also find stresses in bar AB & BC.



Take cross section area for members as 0.2m^2 , $E=220\text{GPa}$.

(6)

2705/535/72/1800